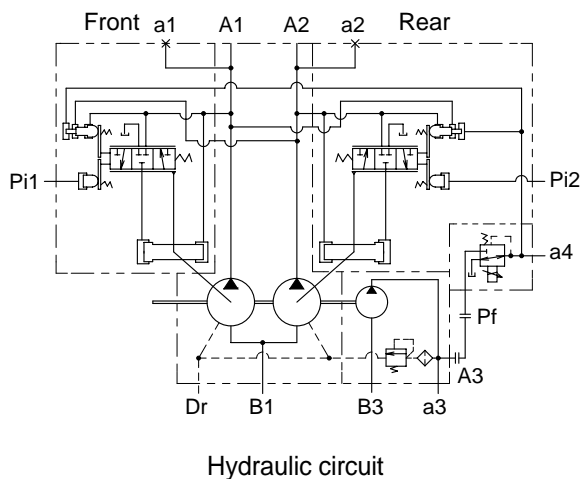
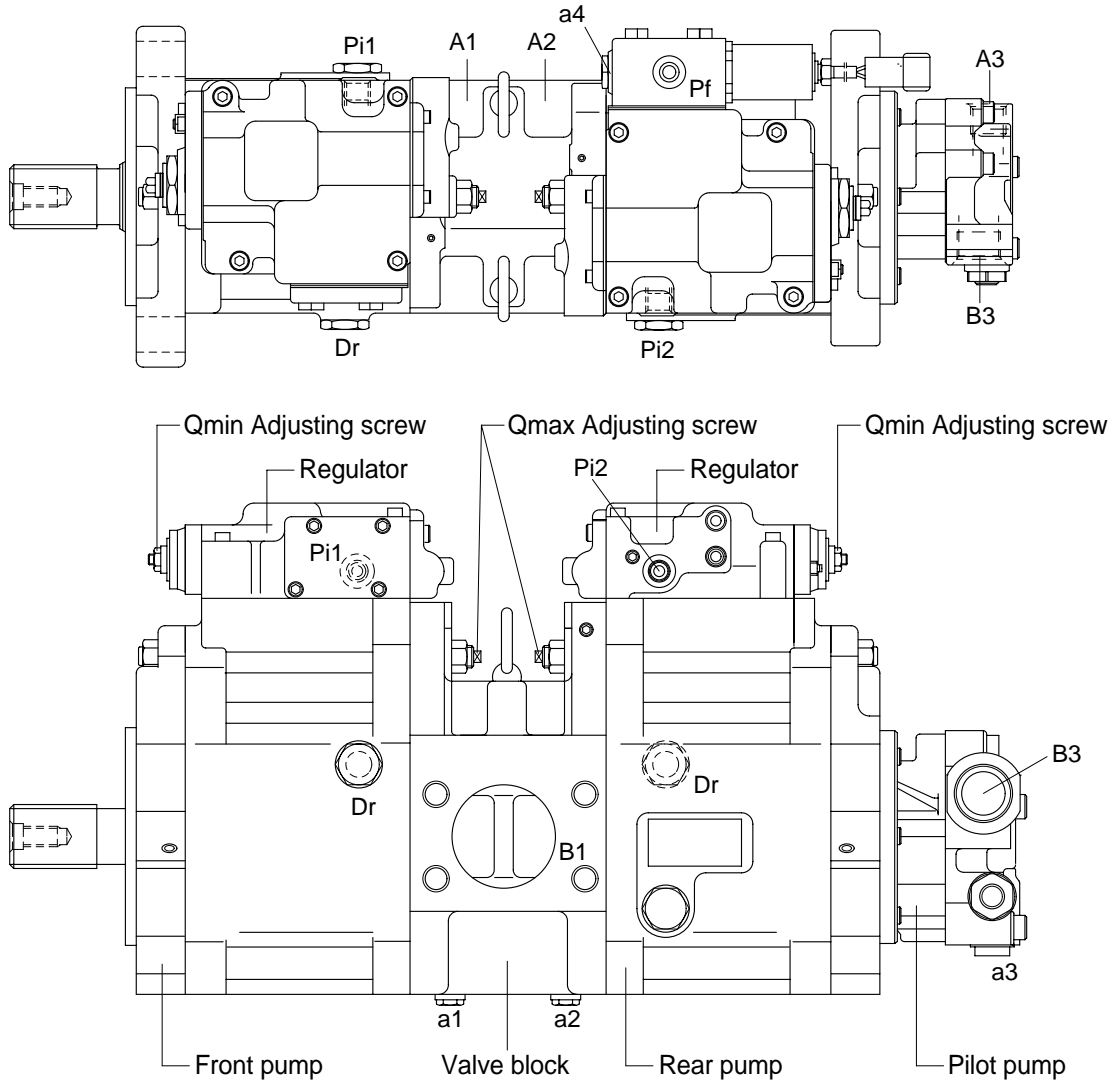


## SECTION 2 STRUCTURE AND FUNCTION

### GROUP 1 PUMP DEVICE

#### 1. STRUCTURE

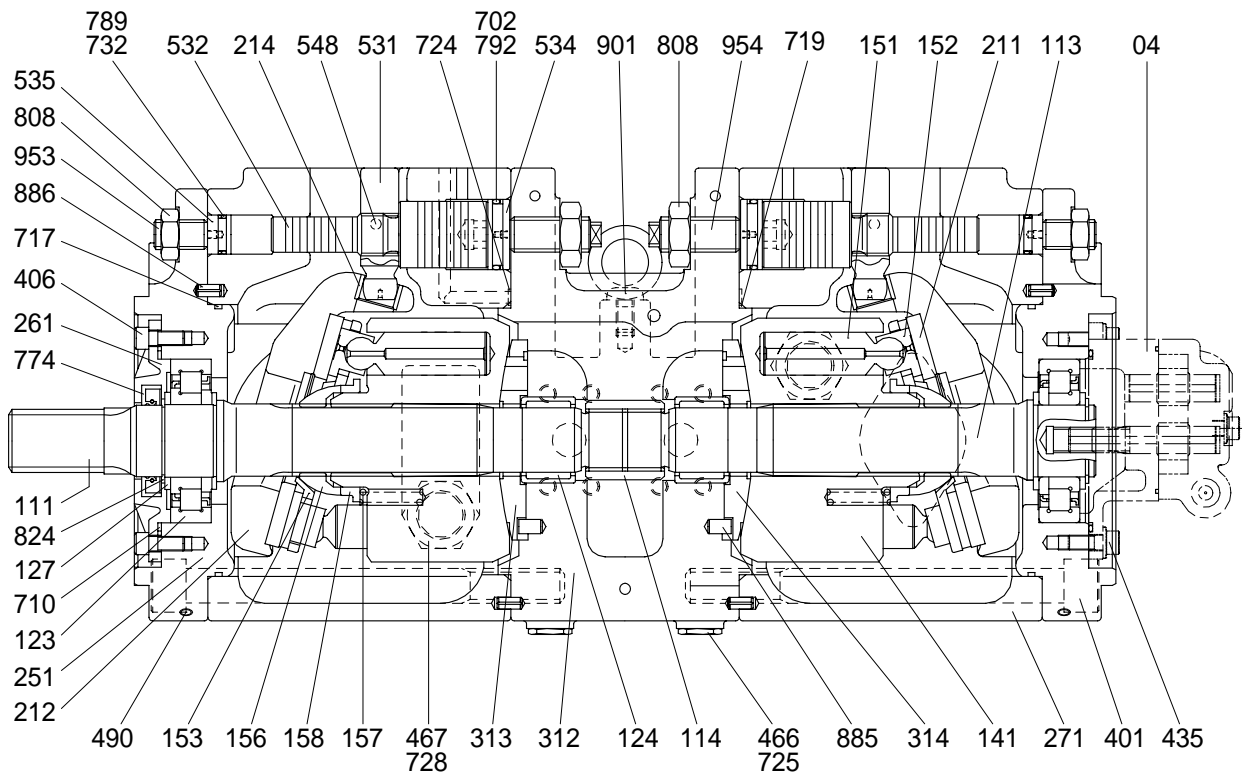
The pump device consists of main pump, regulator and gear pump.



Port	Port name	Port size
A1,2	Delivery port	SAE6000psi 3/4"
B1	Suction port	SAE2500psi 2 1/2"
Dr	Drain port	PF 1/2 - 19
Pf	Power shift port (For decreasing)	PF 1/4 - 15
Pi1,i2	Pilot port	PF 1/4 - 15
a1,2,3,4	Gauge port	PF 1/4 - 15
A3	Gear pump delivery port	PF 1/2 - 19
B3	Gear pump suction port	PF 3/4 - 20.5

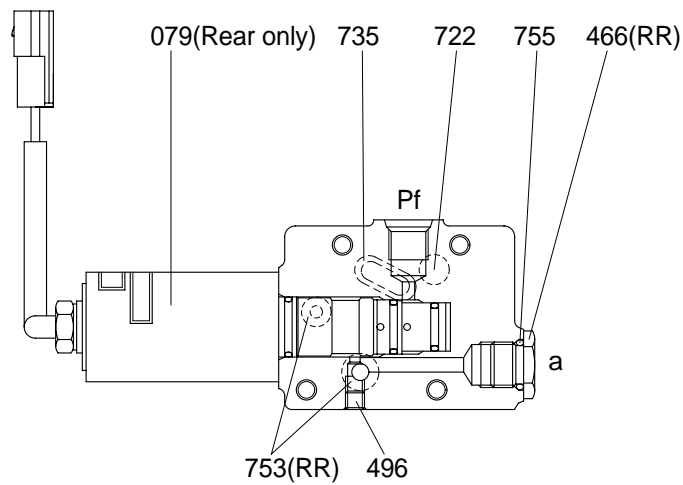
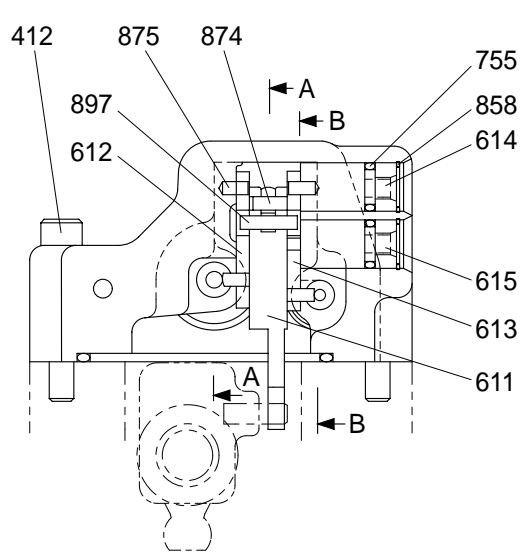
## 1) MAIN PUMP

The main pump consists of two piston pumps(Front & rear) and valve block.

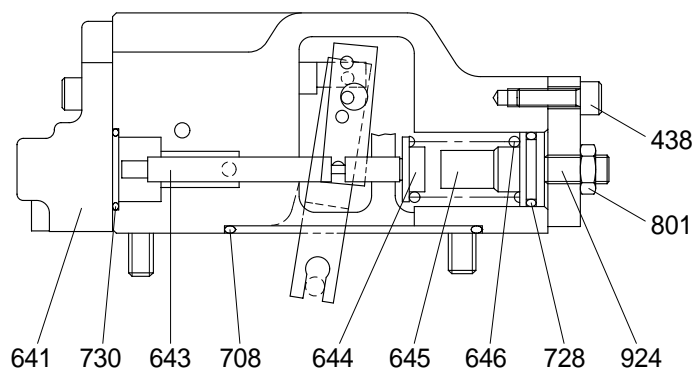


04	Gear pump	261	Seal cover(F)	717	O-ring
111	Drive shaft(F)	271	Pump casing	719	O-ring
113	Drive shaft(R)	312	Valve block	724	O-ring
114	Spline coupling	313	Valve plate(R)	725	O-ring
123	Roller bearing	314	Valve plate(L)	728	O-ring
124	Needle bearing	401	Hexagon socket bolt	732	O-ring
127	Bearing spacer	406	Hexagon socket bolt	774	Oil seal
141	Cylinder block	435	Flange socket bolt	789	Back up ring
151	Piston	466	VP plug	792	Back up ring
152	Shoe	467	VP plug	808	Hexagon head nut
153	Push plate	490	VP plug	824	Snap ring
156	Bushing	531	Tilting pin	885	Pin
157	Cylinder spring	532	Servo piston	886	Spring pin
158	Spacer	534	Stopper(L)	901	Eye bolt
211	Shoe plate	535	Stopper(S)	953	Set screw
212	Swash plate	548	Pin	954	Set screw
214	Bushing	702	O-ring		
251	Swash plate support	710	O-ring		

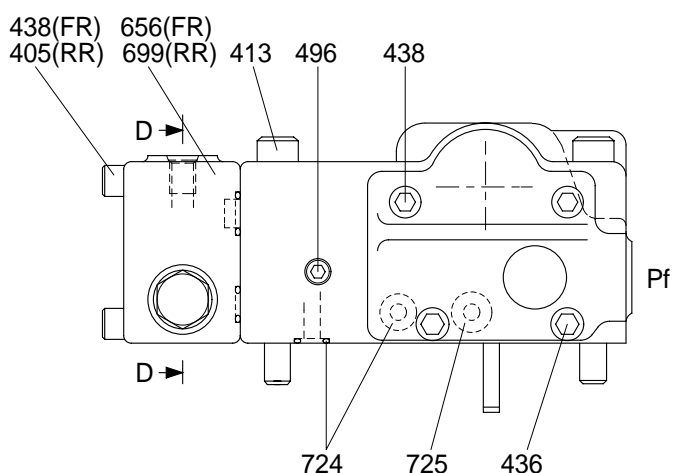
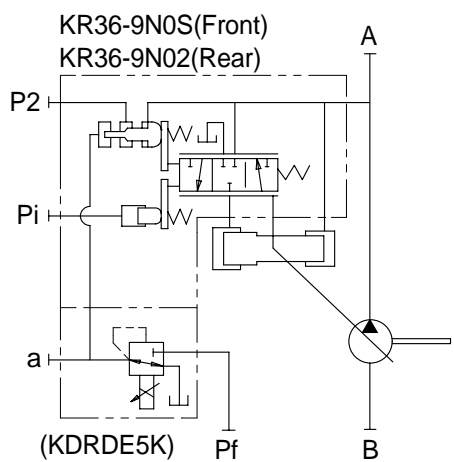
2) REGULATOR(1/2)



SECTION D-D



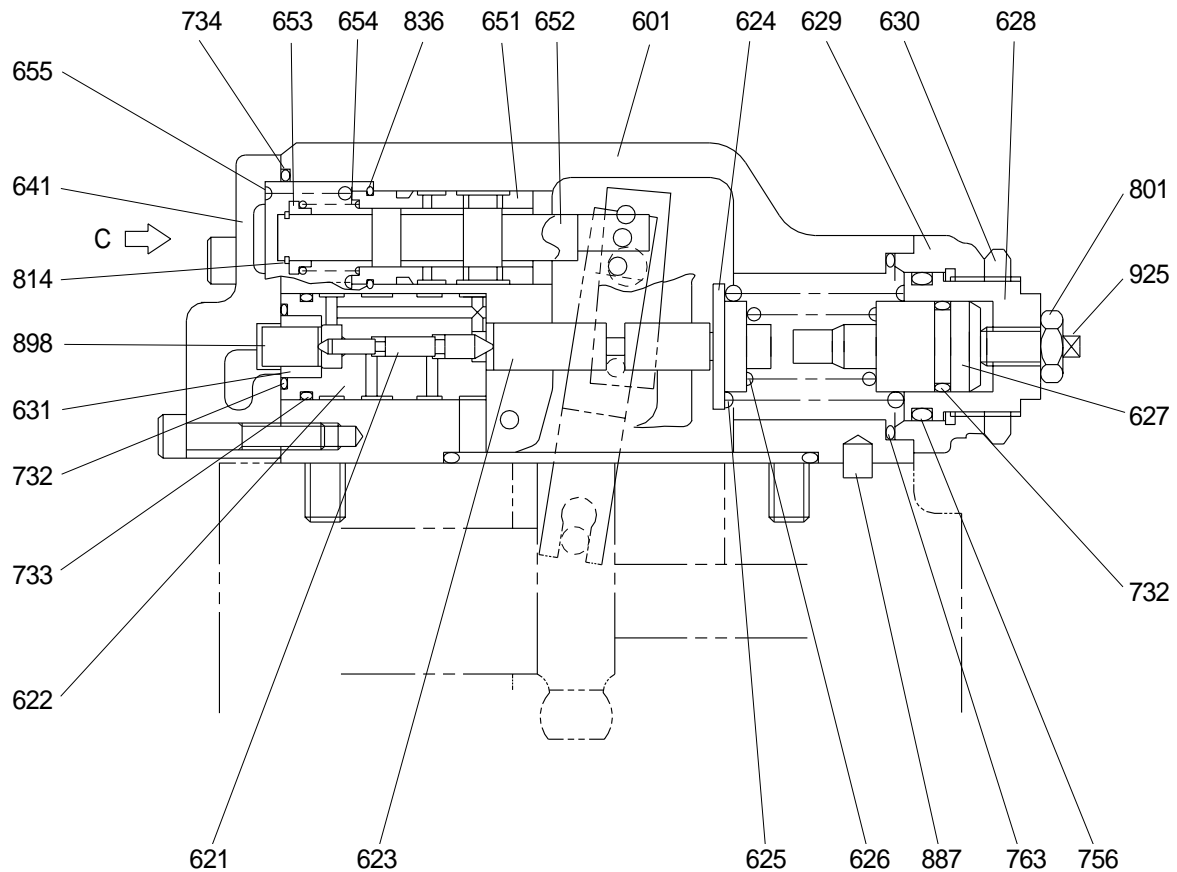
SECTION B-B



VIEW C

Port	Port name	Port size
A	Delivery port	3/4"
B	Suction port	2 1/2"
Pi	Pilot port	PF 1/4-15
Pf	Powershift port	PF 1/4-15
P2	Companion delivery port	-
a	Gauge port	PF 1/4-15

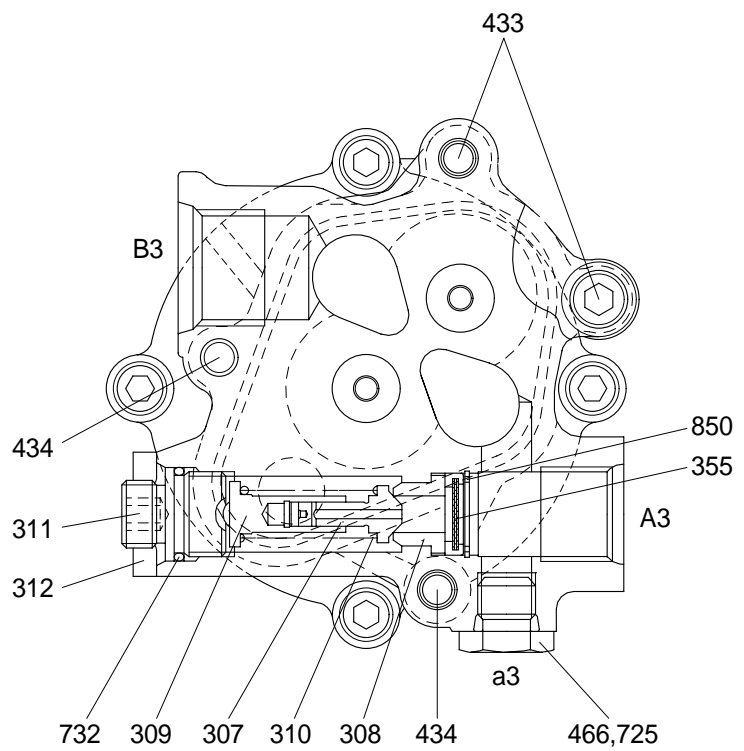
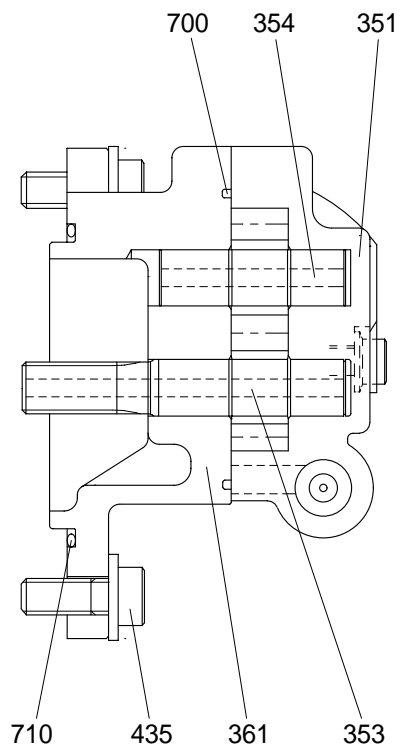
## REGULATOR(2/2)



SECTION A-A

079	EPPR valve assembly	628	Adjust screw(C)	728	O-ring
405	Hexagon socket screw	629	Cover(C)	730	O-ring
412	Hexagon socket screw	630	Nut	732	O-ring
413	Hexagon socket screw	631	Sleeve	733	O-ring
436	Hexagon socket screw	641	Pilot cover	734	O-ring
438	Hexagon socket screw	642	Pilot cover(QMC)	735	O-ring
466	Plug	643	Pilot piston	753	O-ring
496	Plug	644	Spring seat(Q)	755	O-ring
601	Casing	645	Adjust stem(Q)	756	O-ring
611	Feedback lever	646	Pilot spring	763	O-ring
612	Lever(1)	651	Sleeve	801	Nut
613	Lever(2)	652	Spool	814	Snap ring
614	Fulcrum plug	653	Spring seat	836	Snap ring
615	Adjust plug	654	Return spring	858	Snap ring
621	Piston	655	Set spring	874	Pin
622	Piston case	656	Cover	875	Pin
623	Compensator rod	699	Valve casing	887	Pin
624	Spring seat(C)	708	O-ring	897	Pin
625	Outer spring	722	O-ring	898	Pin
626	Inner spring	724	O-ring	924	Set screw
627	Adjust stem(C)	725	O-ring	925	Adjust screw(QI)

### 3) GEAR PUMP



307 Poppet  
 308 Seat  
 309 Spring seat  
 310 Spring  
 311 Adjust screw  
 312 Lock nut  
 351 Gear case

353 Drive gear  
 354 Driven gear  
 355 Filter  
 361 Front case  
 433 Flange socket  
 434 Flange socket  
 435 Flange socket

466 Plug  
 700 Ring  
 710 O-ring  
 725 O-ring  
 732 O-ring  
 850 Snap ring

## 2. FUNCTION

### 1) MAIN PUMP

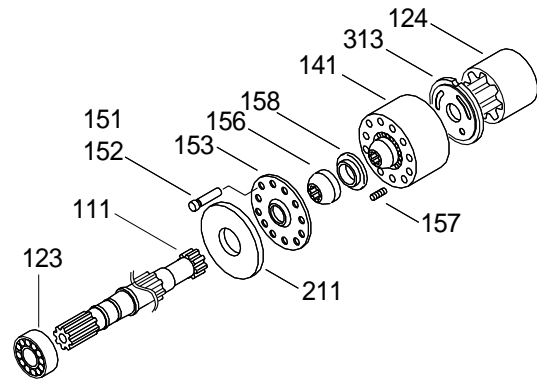
The pumps may be classified roughly into the rotary group performing a rotary motion and working as the major part of the whole pump function : The swash plate group that varies the delivery rates: And the valve cover group that changes over oil suction and discharge.

#### (1) Rotary group

The rotary group consists of drive shaft (F)(111), cylinder block(141), piston shoes(151,152), set plate(153), spherical bush(156), spacer(158) and cylinder spring(157). The drive shaft is supported by bearing(123,124) at its both ends.

The shoe is caulked to the piston to form a spherical coupling. It has a pocket to relieve thrust force generated by loading pressure and the take hydraulic balance so that it slides lightly over the shoe plate(211). The sub group composed by a piston and a shoe is pressed against the shoe plate by the action of the cylinder spring via a retainer and a spherical bush.

Similarly, the cylinder block is pressed against valve plate(313) by the action of the cylinder spring.

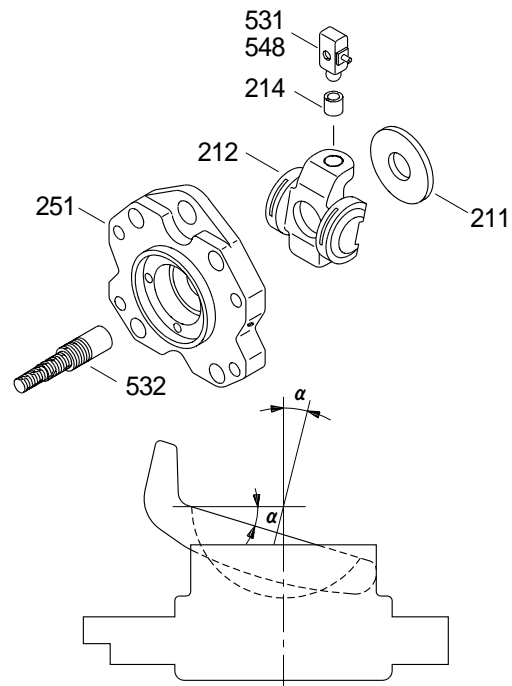


#### (2) Swash plate group

The swash plate group consists of swash plate(212), shoe plate(211), swash plate support(251), tilting bush(214), tilting pin(531) and servo piston(532).

The swash plate is a cylindrical part formed on the opposite side of the sliding surface of the shoe and is supported by the swash support.

If the servo piston moves to the right and left as hydraulic force controlled by the regulator is admitted to hydraulic chamber located on both sides of the servo piston, the swash plate slides over the swash plate support via the spherical part of the tilting pin to change the tilting angle( $\alpha$ )



### (3) Valve block group

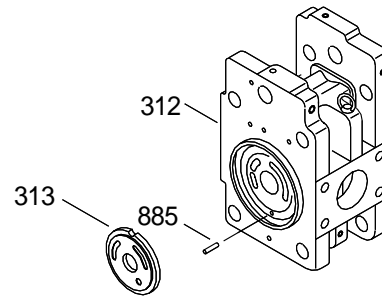
The valve block group consists of valve block(312), valve plate(313) and valve pin(885).

The valve plate having two melon-shaped ports is fixed to the valve block and feeds and collects oil to and from the cylinder block.

The oil changed over by the valve plate is connected to an external pipeline by way of the valve block.

Now, if the drive shaft is driven by a prime mover(Electric motor, engine, etc), it rotates the cylinder block via a spline linkage at the same time. If the swash plate is tilted as in Fig(Previous page) the pistons arranged in the cylinder block make a reciprocating motion with respect to the cylinder block, while they revolve with the cylinder block.

If you pay attention to a single piston, it performs a motion away from the valve plate(Oil sucking process) within 180 degrees, and makes a motion towards the valve plate(Or oil discharging process) in the rest of 180 degrees. When the swash plate has a tilting angle of zero, the piston makes no stroke and discharges no oil.



## 2) REGULATOR

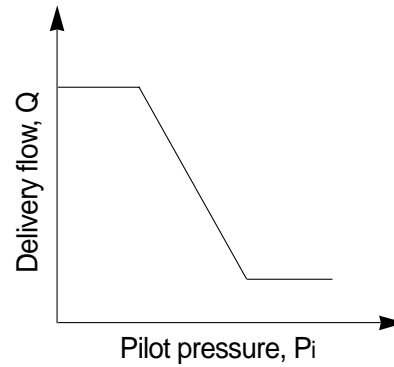
Regulator consists of the negative flow control, total horse power control and power shift control function.

### (1) Negative flow control

By changing the pilot pressure  $P_i$ , the pump tilting angle (Delivery flow) is regulated arbitrarily, as shown in the figure.

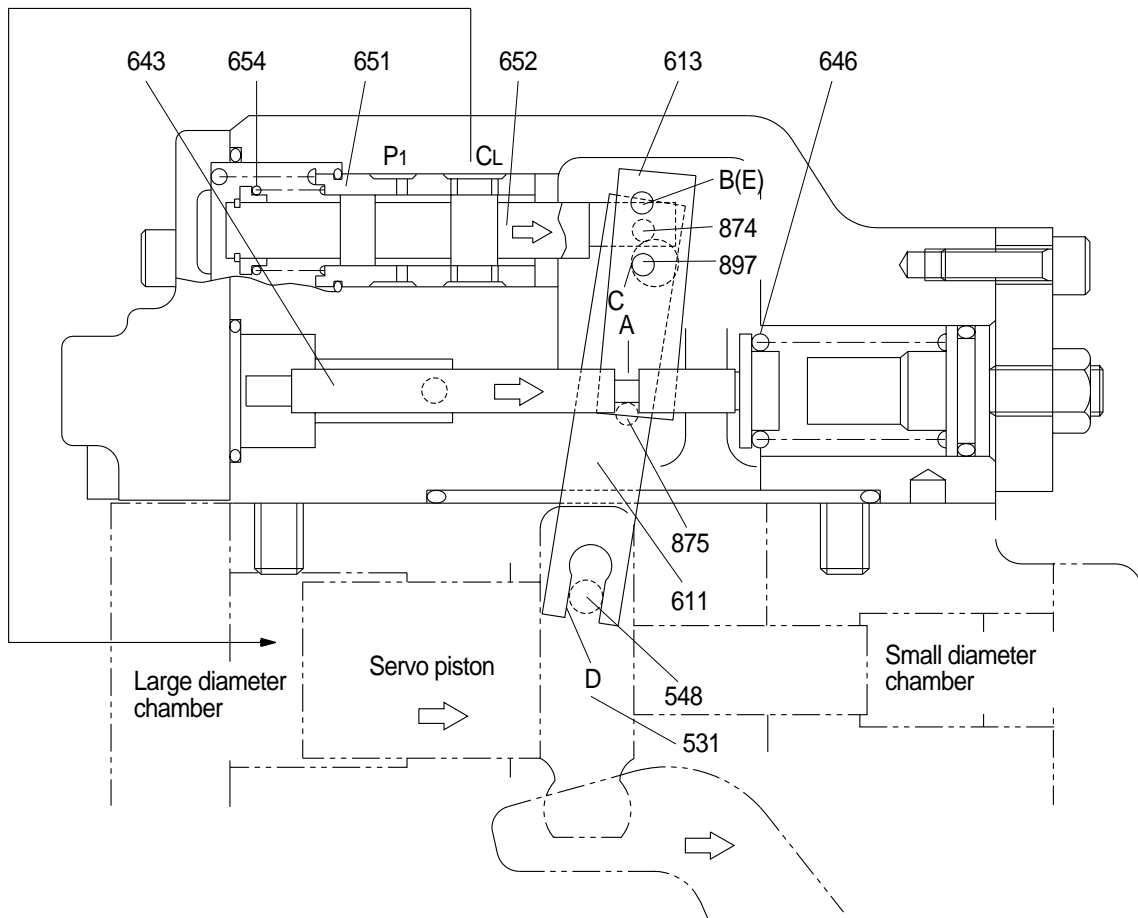
This regulator is of the negative flow control in which the delivery flow  $Q$  decreases as the pilot pressure  $P_i$  rises.

With this mechanism, when the pilot pressure corresponding to the flow required for the work is commanded, the pump discharges the required flow only, and so it does not consume the power uselessly.





## ① Flow reducing function



As the pilot pressure  $P_i$  rises, the pilot piston(643) moves to the right to a position where the force of the pilot spring(646) balances with the hydraulic force.

The groove(A) in the pilot piston is fitted with the pin(875) that is fixed to lever 2(613). Therefore, when the pilot piston moves, lever 2 rotates around the fulcrum of point B [fixed by the fulcrum plug(614) and pin(875)]. Since the large hole section(C) of lever 2 contains a protruding pin(897) fixed to the feedback lever(611), the pin(897) moves to the right as lever 2 rotates. Since the opposing-flat section(D) of the feedback lever is fitted with the pin(548) fixed by the tilting pin(531) that swings the swash plate, the feedback lever rotates around the fulcrum of point D, as the pin(897) moves.

Since the feedback lever is connected with the spool(652) via the pin(874), the spool moves to the right.

The movement of the spool causes the delivery pressure  $P_1$  to connect to port CL through the spool and to be admitted to the large diameter section of the servo piston. The delivery pressure  $P_1$  that is constantly admitted to the small diameter section of the servo piston moves the servo piston to the right due to the area difference, resulting in decrease of the tilting angle.

When the servo piston moves to the right, point D also moves to the right. The spool is fitted with the return spring(654) and is tensioned to the left at all times, and so the pin(897) is pressed against the large hole section(C) of lever 2.

Therefore, as point D moves, the feedback lever rotates around the fulcrum of point C, and the spool is shifted to the left. This causes the opening between the sleeve(651) and spool(652) to close slowly, and the servo piston comes to a complete stop when it closes completely.

As the servo piston moves, point D also moves to the left, the feedback lever rotates around the fulcrum of point C, and the spool moves to the right till the opening between the spool and sleeve is closed.

### ③ Adjustment of flow control characteristic

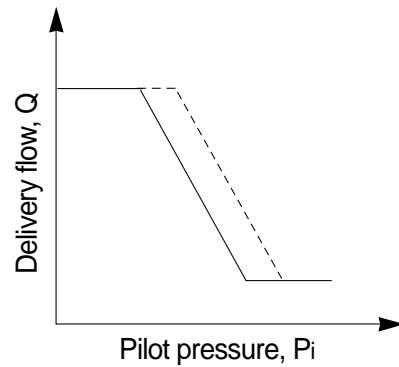
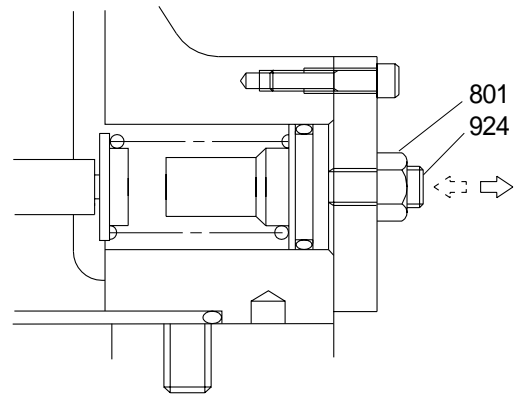
The flow control characteristic can be adjusted with the adjusting screw.

Adjust it by loosening the hexagon nut (801) and by tightening(Or loosening) the hexagonal socket head screw(924).

Tightening the screw shifts the control chart to the right as shown in the figure.

※ Adjusting values are shown in table

Speed	Adjustment of flow control characteristic		
	Tightening amount of adjusting screw(924)	Flow control starting pressure change amount	Flow change amount
(min <sup>-1</sup> )	(Turn)	(kgf/cm <sup>2</sup> )	( l /min)
2200	+1/4	+1.7	+8.2



## (2) Total horsepower control

The regulator decreases the pump tilting angle(Delivery flow) automatically to limit the input torque within a certain value with a rise in the delivery pressure P1 of the self pump and the delivery pressure P2 of the companion pump.

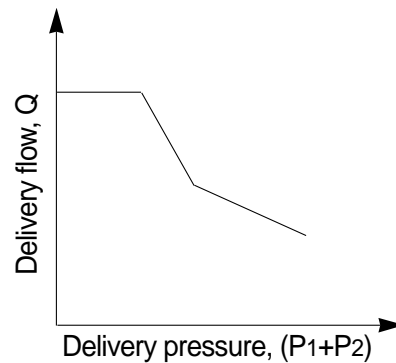
(The input horsepower is constant when the speed is constant.)

Since the regulator is of the simultaneous total horsepower type that operates by the sum of load pressures of the two pumps in the tandem double-pump system, the prime mover is automatically prevented from being overloaded, irrespective of the load condition of the two pumps, when horsepower control is under way.

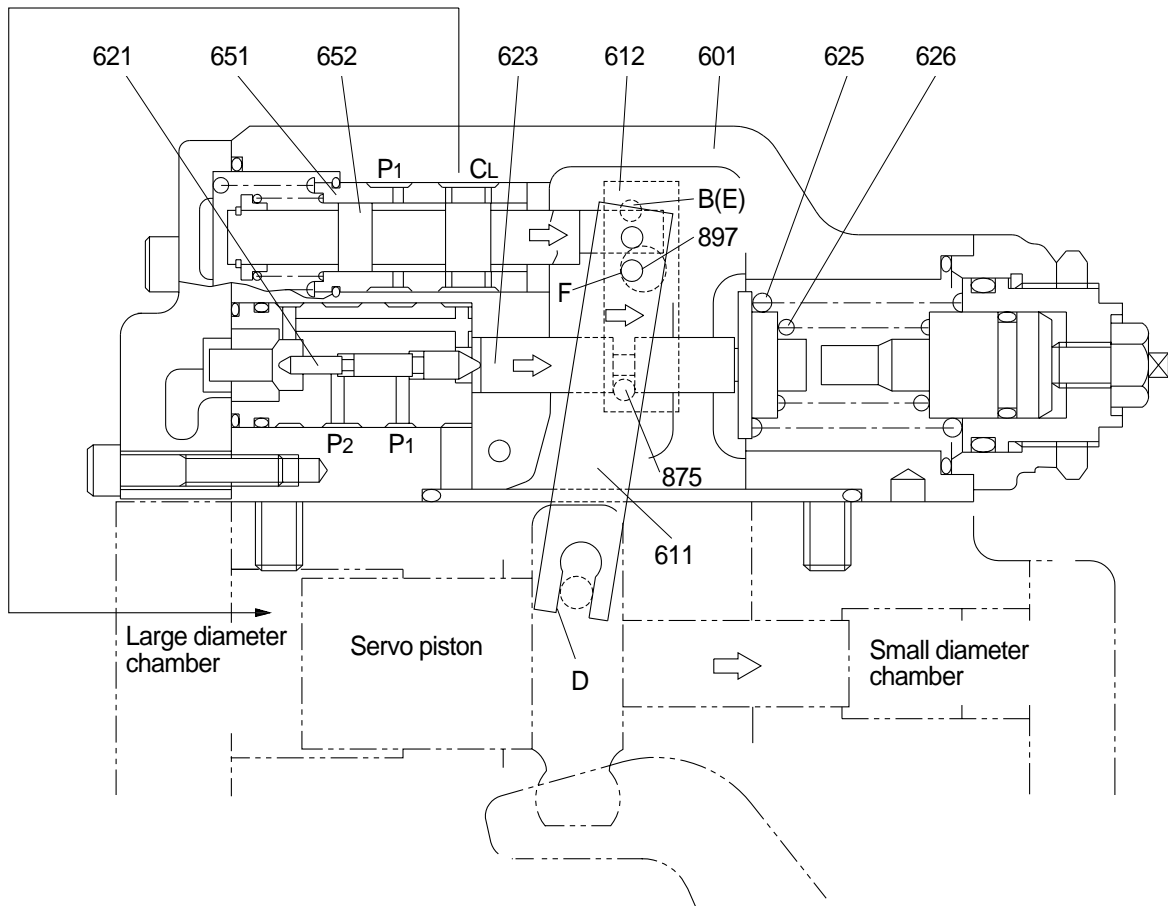
Since this regulator is of the simultaneous total horsepower type, it controls the tilting angles(Displacement volumes) of the two pumps to the same value as represented by the following equation :

$$\begin{aligned} T_{in} &= P1 \times q/2\pi + P2 \times q/2\pi \\ &= (P1+P2) \times q/2\pi \end{aligned}$$

The horsepower control function is the same as the flow control function and is summarized in the following.(For detailed behaviors of respective parts, refer to the section of flow control.)



## ① Overload preventive function



When the self pump delivery pressure P1 or the companion pump delivery pressure P2 rises, it acts on the stepped part of the compensating piston(621). It presses the compensating rod(623) to the right till the force of the outer spring(625) and inner spring(626) balances with the hydraulic force. The movement of the compensating rod is transmitted to lever 1(612) via pin(875).

Lever 1 rotates around the pin(875) (E) fixed to the casing(601).

Since the large hole section(F) of lever 1 contains a protruding pin(897) fixed to the feedback lever(611), the feedback lever rotates around the fulcrum of point D as lever 1 rotates, and then the spool(652) is shifted to the right. As the spool moves, the delivery pressure P1 is admitted to the large diameter section of the servo piston via port CL, causes the servo piston move to the right, reduces the pump delivery, flow rate, and prevents the prime mover from being overloaded. The movement of the servo piston is transmitted to the feedback lever via point D. Then the feedback lever rotates around the fulcrum of point F and the spool is shifted to the left. The spool moves till the opening between the spool(652) and sleeve(651) is closed.

This causes the servo piston to move to the left and the pump's delivery rate to increase. The movement of the servo piston is transmitted to the spool by the action of the feedback mechanism to move it till the opening between the spool and sleeve is closed.

### ③ Low tilting angle(Low flow) command preferential function

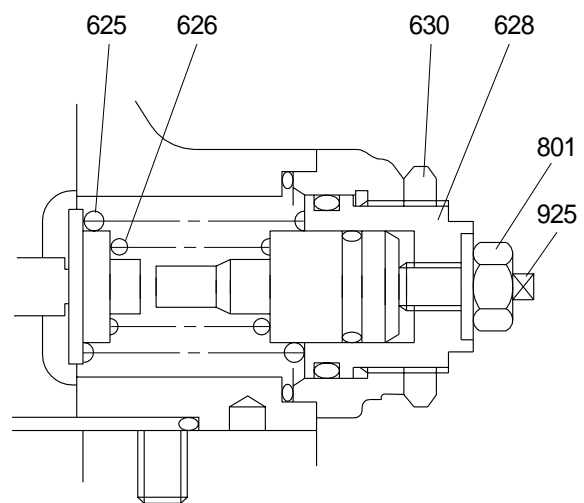
As mentioned above, flow control and horsepower control tilting angle commands are transmitted to the feedback lever and spool via the large-hole sections(C & F) of levers 1 and 2. However, since sections C and F have the pins( $\phi 4$ ) protruding from the large hole( $\phi 8$ ), only the lever lessening the tilting angle contacts the pin(897); The hole( $\phi 8$ ) in the lever of a larger tilting angle command is freed without contacting the pin(897). Such a mechanical selection method permits preference of the lower tilting angle command of the flow control and horsepower control.

### ④ Adjustment of input horsepower

Since the regulator is of total cumulative horsepower type, adjust the adjusting screws of both the front and rear pumps, when changing the horsepower set values. The pressure change values by adjustment are based on two pumps pressurized at the same time, and the values will be doubled when only one pump is loaded.

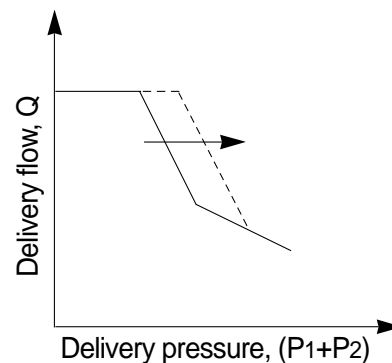
#### a. Adjustment of outer spring

Adjust it by loosening the hexagon nut(630) and by tightening(Or loosening) the adjusting screw C(628). Tightening the screw shifts the control chart to the right and increases the input horsepower as shown in the figure. Since turning the adjusting screw C by N turns changes the setting of the inner spring(626), return the adjusting screw QI(925) by  $N \times A$  turns at first. ( $A=2.2$ )



※ Adjusting values are shown in table

Speed	Adjustment of outer spring		
	Tightening amount of adjusting screw(C) (628)	Compensating control starting pressure change amount	Input torque change amount
(min <sup>-1</sup> )	(Turn)	(kgf/cm <sup>2</sup> )	(kgf · m)
2200	+1/4	+19.2	+3

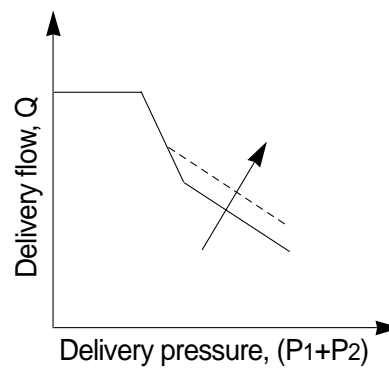
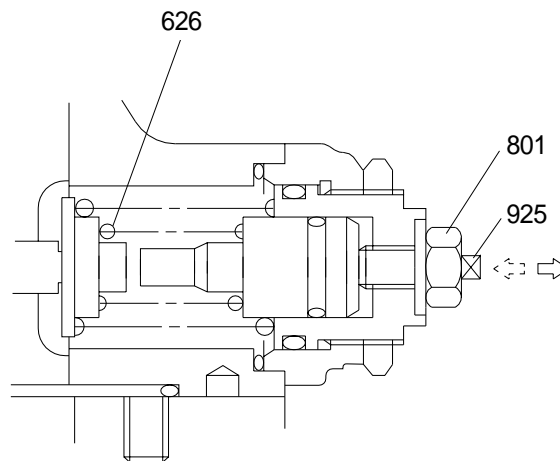


**b. Adjustment of inner spring**

Adjust it by loosening the hexagon nut (801) and by tightening (Or loosening) the adjusting screw QI (925). Tightening the screw increases the flow and then the input horsepower as shown in the figure.

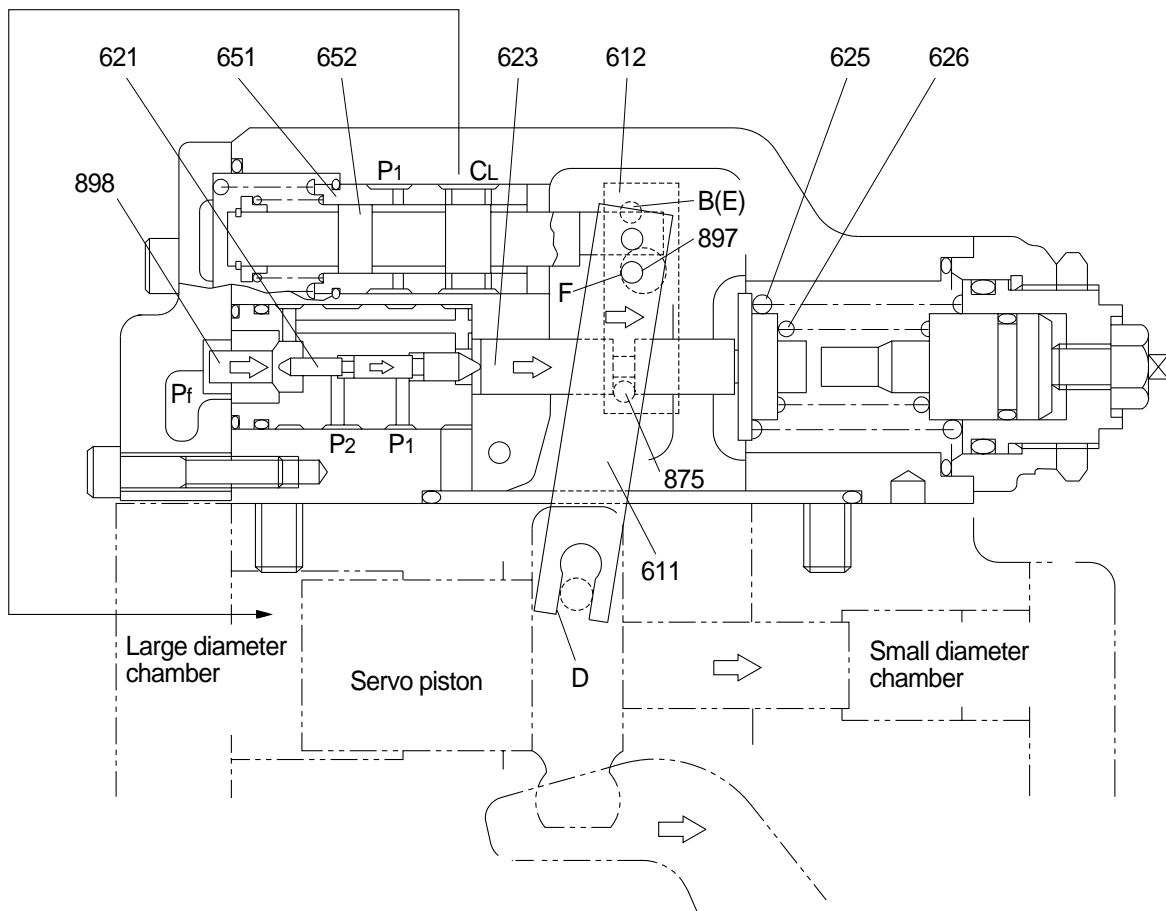
※ Adjusting valves are shown in table

Speed	Adjustment of inner spring		
	Tightening amount of adjusting screw (QI) (925)	Compensating control starting pressure change amount	Input torque change amount
(min <sup>-1</sup> )	(Turn)	(kgf/cm <sup>2</sup> )	(kgf · m)
2200	+1/4	+27.3	+2.5





### (3) Power shift control



The set horsepower valve is shifted by varying the command current level of the proportional pressure reducing valve attached to the pump.

Only one proportional pressure reducing valve is provided.

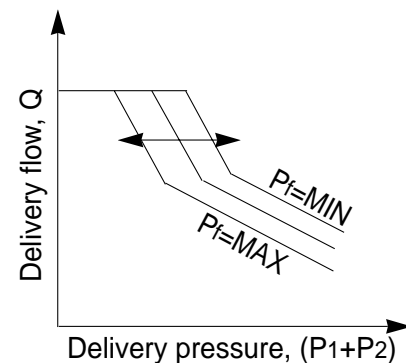
However, the secondary pressure  $P_f$  (Power shift pressure) is admitted to the horsepower control section of each pump regulator through the pump's internal path to shift it to the same set horsepower level.

This function permits arbitrary setting of the pump output power, thereby providing the optimum power level according to the operating condition.

The power shift pressure  $P_f$  controls the set horsepower of the pump to a desired level, as shown in the figure.

As the power shift pressure  $P_f$  rises, the compensating rod(623) moves to the right via the pin(898) and compensating piston(621).

This decreases the pump tilting angle and then the set horsepower in the same way as explained in the overload preventive function of the horsepower control. On the contrary, the set horsepower rises as the power shift pressure  $P_f$  falls.



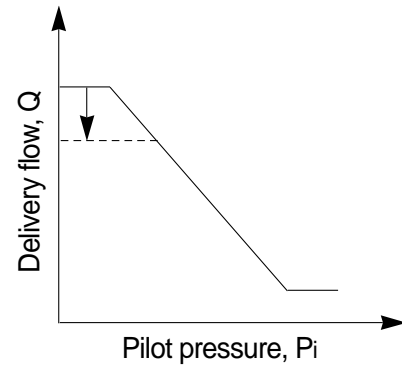
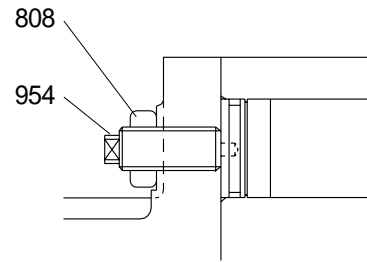
#### (4) Adjustment of maximum and minimum flows

##### ① Adjustment of maximum flow

Adjust it by loosening the hexagon nut(808) and by tightening(Or loosening) the set screw(954).

The maximum flow only is adjusted without changing other control characteristics.

Speed	Adjustment of max flow	
	Tightening amount of adjusting screw (954)	Flow change amount
(min <sup>-1</sup> )	(Turn)	( l /min)
2200	+1/4	-3.2



##### ② Adjustment of minimum flow

Adjust it by loosening the hexagon nut (808) and by tightening(Or loosening) the hexagonal socket head set screw (953). Similarly to the adjustment of the maximum flow, other characteristics are not changed.

However, remember that, if tightened too much, the required horsepower during the maximum delivery pressure(Or during relieving) may increase.

Speed	Adjustment of min flow	
	Tightening amount of adjusting screw (953)	Flow change amount
(min <sup>-1</sup> )	(Turn)	( l /min)
2200	+1/4	+3.2

